MATH 215/255, SECTION 102, MIDTERM II, NOVEMBER 19 2012 (H. DIXIT) Closed Book and Notes. Calculators not allowed. 50 Minutes. Total: 40 marks This exam contains two pages along with a table of Laplace transforms.

Problem 1: [12 marks] Short answers. Detailed explanation not required. 3 marks for each question.

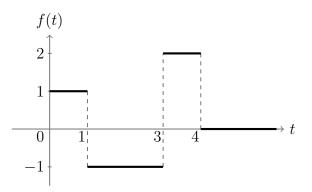
i) Find the particular solution of the non-homogeneous equation

$$y'' + y' + 3y = t.$$

ii) Find the damping constant, γ , such that the following spring-mass-damper system is critically damped:

$$2y'' + \gamma y' + 8y = 0.$$

iii) Express the function, f(t), shown in the figure below in terms of unit step functions.



iv) Find the inverse Laplace transform of

$$F(s) = \frac{2}{s^2 + 3s - 4}$$

Problem 2: [10 points] If $y_1(t) = (1 + t)$ and $y_2(t) = e^t$ are solutions of the homogeneous part of the following differential equation,

$$ty'' - (1+t)y' + y = t^2 e^{2t}, \qquad t > 0,$$

find the particular solution using variation of parameters.

Problem 3: [10 points] Consider a spring-mass-damper system forced by an external forcing with frequency ω . The position of the mass is described by the following initial value problem:

 $u'' + 5u' + 6u = 3\cos(\omega t), \quad u(0) = 1, u'(0) = 0.$

i) Determine the steady state or forced response of the system. [7 points]

ii) The steady state response can be expressed as $R\cos(\omega t - \delta)$. Determine the amplitude R in terms of ω . [3 points]

Problem 4: [8 points] Consider the following differential equation with discontinuous forcing:

$$y'' + 3y' + 2y = \begin{cases} 1 & 0 \le t < 10, \\ 0 & t \ge 10, \end{cases}; \quad y(0) = y'(0) = 0,$$

i) Calculate the solution y(t) using Laplace transforms. [6 points]

ii) Determine $\lim_{t\to\infty} y(t)$. [2 points]

Table of Elementary Laplace Transforms			
		$f(t) = \mathcal{L}^{-1}\{F(s)\}$	$F(s) = \mathcal{L}\{f(t)\}$
	1.	1	$\frac{1}{s}, s > 0$
	2.	e^{at}	$\frac{1}{s-a}, s > a$
	3.	t^n , $n = $ positive integer	$\frac{n!}{s^{n+1}}, s > 0$
	4.	$t^p, p > -1$	$\frac{\Gamma(p+1)}{s^{p+1}}, s > 0$
	5.	$\sin(at)$	$\frac{a}{s^2 + a^2}, s > 0$
	6.	$\cos(at)$	$\frac{s}{s^2 + a^2}, s > 0$
	7.	$\sinh(at)$	$\frac{a}{s^2 - a^2}, s > a $
	8.	$\cosh(at)$	$\frac{s}{s^2 - a^2}, s > a $
	9.	$e^{at}\sin(bt)$	$\frac{b}{(s-a)^2+b^2}, s>a$
	10.	$e^{at}\cos(bt)$	$\frac{s-a}{(s-a)^2+b^2}, s>a$
	11.	$t^n e^{at}$, $n = \text{positive integer}$	$\frac{n!}{(s-a)^{n+1}}, s > a$
	12.	$u_c(t)$	$\frac{e^{-cs}}{s}, s > 0$
	13.	$u_c(t)f(t-c)$	$e^{-cs}F(s)$
	14.	$e^{ct}f(t)$	F(s-c)
	15.	f(ct)	$\frac{1}{c}F\left(\frac{s}{c}\right), c > 0$
	16.	$\int_0^t f(t-\tau)g(\tau)d\tau$	F(s)G(s)
	17.	$\delta(t-c)$	e^{-cs}
	18.	$f^{(n)}(t)$	$s^{n}F(s) - s^{n-1}f(0) - \dots - f^{(n-1)}(0)$
	19.	$(-1)^n f(t)$	$F^{(n)}(s)$

Table of Elementary Laplace Transforms

MATH 215/255 MIDTERM I SOLUTIONS

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Problem 1: y'' + y' + 3y = t(ټ) Find portreular solution. Using method of undetermined coefficients, let $y_p = At+B$ =) yp'= A $y_{\rho}^{"} = 0$ $=) \quad 0 + A + 3(At+B) = t$ & A+3B=0 : $y_{p} = \frac{1}{3} - \frac{1}{9}$ Find I such that the System is critically damped. (11) $2\lambda^2 + \sqrt[3]{\lambda} + 8 = 0$ charactustic equation: =) $\lambda = -\frac{1}{2} \sqrt{\frac{1}{1}} \sqrt{\frac{1}{1}} \sqrt{\frac{1}{2}} \sqrt{\frac{1}{2}} \sqrt{\frac{1}{2}} \sqrt{\frac{1}{2}}$ $= -r \pm \sqrt{r^2 - 64}$ when $\gamma^2 = 64 = 0 \implies \gamma = 8$ withally domped are complete conjugates: Underdamped System If 1<8, 2's real 4 negative : Over damped. Note ! ij 178, 2's are

$$= 1 + (-2)U_{1}(t) + 3U_{3}(t) - 2U_{4}(t)$$
$$= 1 - 2U_{1}(t) + 3U_{3}(t) - 2U_{4}(t)$$

(iv)
$$F(S) = \frac{2}{S^{2}+3S-4}$$

 $= \frac{2}{(S+4)(S-1)} = R\left[\frac{A}{S+4} + \frac{B}{S-1}\right]$
 $\Rightarrow A(S-1) + B(S+4) = 1 \Rightarrow -A+B=0 \quad Y \quad A = -\frac{1}{5}$
 $\therefore F(S) = \frac{2}{5}\left[\frac{-1}{5+4} + \frac{1}{5-1}\right]$

=)
$$f(t) = \frac{2}{5} \left[-e^{-4t} + e^{t} \right]$$

$$\begin{array}{rcl} P_{Ao} \underline{b} \underline{b} \underline{h} & 2 \\ & ty^{ii} - (it+1) y^{i} + y = t^{2} e^{2t} &, t \neq 0 \\ & y_{i}(t) = it^{i} & y_{i}(t) = t^{i} & y_{i}(t) \\ & y_{i}(t) = e^{t} & y_{i}(t) + ty = t^{2} e^{2t} \\ \Rightarrow & y_{i}(t) = e^{t} & y_{i}(t) + ty = t^{2} e^{2t} \\ \Rightarrow & y_{i}(t) = (it+1) + ty = t^{2} e^{t} \\ \Rightarrow & y_{i}(t) = (it+1) + ty = t^{2} e^{t} \\ \Rightarrow & y_{i}(t) = (it+1) + ty = t^{2} e^{t} \\ & y_{i}(t) = (it+1) + ty = t^{2} e^{t} \\ & y_{i}(t) = (it+1) + ty = t^{2} e^{t} \\ & y_{i}(t) = (it+1) + ty = t^{2} e^{t} \\ & y_{i}(t) = (it+1) + ty = t^{2} e^{t} \\ & y_{i}(t) = (it+1) + ty = t^{2} e^{t} \\ & y_{i}(t) = (it+1) + ty = t^{2} e^{t} \\ & y_{i}(t) = (it+1) + ty = t^{2} e^{t} \\ & y_{i}(t) = (it+1) + ty = t^{2} e^{t} \\ & y_{i}(t) = (it+1) + ty = t^{2} e^{t} \\ & y_{i}(t) = (it+1) + ty = t^{2} e^{t} \\ & y_{i}(t) = t^{2} e^{t} \\ & y_{i}(t) = (it+1) + ty = t^{2} e^{t} \\ & y_{i}(t) = t^{2} e^{t} \\ &$$

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$$u_1 \left[-(1+t) + (1+t) \right] + u_2 \left[-(1+t)e^t + te^t + e^t \right]$$

+ $tu_1' + u_2' te^t = t^2 e^{2t}$

$$\begin{bmatrix} 4i'\\ u_2 \end{bmatrix} = \underbrace{ \begin{bmatrix} e^{t} & -e^{t} \end{bmatrix} }_{U+t} \begin{bmatrix} 0\\ te^{2t} \end{bmatrix}$$

$$= \frac{1}{te^{t}} \begin{bmatrix} e^{t} & -e^{t} \end{bmatrix} \begin{bmatrix} 0\\ te^{2t} \end{bmatrix}$$

=)
$$u_1' = \frac{1}{te^t} \times -te^{3t} = -e^{3t} - e^{2t}$$

 $A \quad u_2' = \frac{1}{te^t} \times (1+t) te^{2t} = (1+t) e^t$

$$\therefore \quad \forall u_1 = -\frac{e}{2} + c_1$$

$$\overrightarrow{v_1} = -\frac{e}{2} + c_1$$

$$\overrightarrow{v_2} = te^{t} + c_2$$

$$\therefore y_{p} = \left(-\frac{e^{2t}}{2} + 4\right)(1+t) + \left(te^{t} + 1\right)e^{t}$$

$$= \left(\frac{1+t}{2} + 1\right) + \left(1+t\right) + \left(te^{t} - \frac{1+t}{2}\right)e^{2t} + te^{2t}$$

$$\int Same \quad a_{3}$$

$$\int Same \quad a_{3}$$

$$\int Same \quad a_{4}$$

$$\int Same$$

$$\frac{P_{10}blum(3)}{U'' + 5U' + bU} = 3 \log (wt) ; U(0) = 1 U'(0) = 0.$$

(1) Steady state response = Postitulal Solution.
Using method of underwand coefficients, let

$$U_p = A \cos(\omega t) + B \sin(\omega t)$$

 $U_p' = -A \omega \sin(\omega t) + B \omega \cos(\omega t)$
 $U_p'' = -A \omega \sin(\omega t) - B \omega^2 \sin(\omega t)$
 $U_p'' = -A \omega^2 \cos(\omega t) - B \omega^2 \sin(\omega t)$

Substituting, we have

$$\int -Aw^{2} (os |wt) - Bw^{2} (in (wt)] + 5 \left[-Aw (in (wt)) + Bw (os (wt)) \right]$$

$$+ 6 \left[A (os (wt)) + B (in (wt)) \right] = 3 (os (wt))$$

$$O(\log (\omega t)): -A\omega^{2} + 5B\omega + 6A = 3 \rightarrow (6-\omega^{2})B = 5A\omega$$
$$O(\sin \omega t): -B\omega^{2} - 5A\omega + 6B = 0 \longrightarrow (6-\omega^{2})B = 5A\omega$$
$$=) A = \frac{6-\omega^{2}}{5\omega}B$$

$$=) (6-\omega^{2}) \frac{6-\omega^{2}}{5\omega} B + 5\omega B = 3$$

$$=) [(6-\omega^{2})^{2} + 25\omega^{2}] B = 15\omega$$

$$=) B = \frac{15\omega}{[(6-\omega^{2})^{2} + 25\omega^{2}]}$$

$$\therefore A = \frac{3(6-\omega^{2})}{[(6-\omega^{2})^{2} + 25\omega^{2}]}$$

. Steady state respond:

$$U_p = \frac{3(6-w^2)}{(6-w^2)^2} + 25w^2$$

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(i) Let
$$U_p = R \log (\omega t - d)$$

= $R \log d \cos (\omega t) + R \ln d \cdot \sin (\omega t)$

$$=) \quad R^{2} = A^{2} + 6^{2}$$

$$= \frac{9(6 - 10^{2})^{2}}{\left[(6 - 10^{2})^{2} + 256^{2}\right]^{2}} + \frac{2256^{2}}{\left[(6 - 10^{2})^{2} + 256^{2}\right]}$$

$$= \frac{9\left[\left(6 - 62^{2}\right)^{2} + 256^{2}\right]}{\left[(6 - 62^{2})^{2} + 256^{2}\right]^{2}}$$

$$R = \frac{3}{\sqrt{(6-w^2)^2 + 25w^2}}$$

$$\int \frac{1}{(6-w^2)^2 + 25w^2} = \frac{3}{\sqrt{(6-w^2)^2 + 25w^2}}$$

$$\int \frac{1}{\sqrt{(6-w^2)^2 + 25w^2}} = \frac{3}{\sqrt{(6-w^2)^2 + 25w^2}}$$

Note that if the damping

blomes

$$u^{11} + 6u = 3 \log(\omega t)$$

solving for amplitude R, we get
 $R = \frac{3}{(6-\omega^2)}$
Resonance
 $R = \sqrt{6}$, $R \to \infty$;

Problem (1):
$$y'' + 3y' + 2y = \begin{cases} 1 & 0 \le t < 10 \\ 0 & t = 7,10 \end{cases}$$

(1) Taking Loplan transform, we have

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=) $y(t) = (2 - e^{-t} - 2e^{-2t}) - 410(t) [2]$

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As $t \rightarrow \infty$; $e^{-t} \rightarrow 0$ $e^{-2t} \rightarrow 0$ A $U_{10}(t) = 1$ $\Rightarrow L_{im} y(t) = \frac{1}{2} - (x[\frac{1}{2}]) = 0$ $t \rightarrow \infty$

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